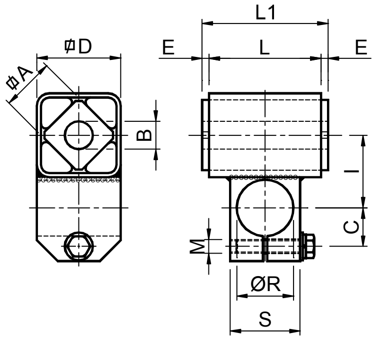
VIB 弹性组件 型号: GF / *Elastic Components* VIB *Type:* GF



型号 *Type*	编号 N°	Q			n	Md	A	B	C	D	E	I	L	L1	M	R	S	重量 *Weight* in kg
		J=2	J=3	J=4														
GF 40	RE021076	280	230	190	570	2.5	27	16	21.5	45	2.5	39	60	65	M10	30	40	0.90
GF 50	RE021078	580	480	380	490	6.4	38	20	26.5	60	5	52	80	90	M10	40	50	1.40

Q: 每个悬架最大负载 以 N 表示/ *Max loading in N per suspension*

J: 振动机器指数/ *Oscillating machine factor*

n: 偏心轮最高旋转速度 以 min^{-1} 表示 最大角度为∢10° 从位置 0 波动∢ ±5°
Max crank rotation velocity in min^{-1} *at the max angle ∢10° from 0 ∢ ±5°*

D_m: 最大振幅 以 mm 表示 / *Max amplitude in mm*

E_d: 动力弹性 以 Nm/°表示 角度为∢ ±5°, 频率范围从 300 至 600 min^{-1}
Dynamic spring value in Nm/° at per ∢ ±5°, in frequency range 300-600 min^{-1}

材料 / MATERIALS

外部结构为钢制，内部方管为铝制拉丝。/ *The external body is made of steel while the inner square is made of light alloy profile.*

处理 / *TREATMENTS*

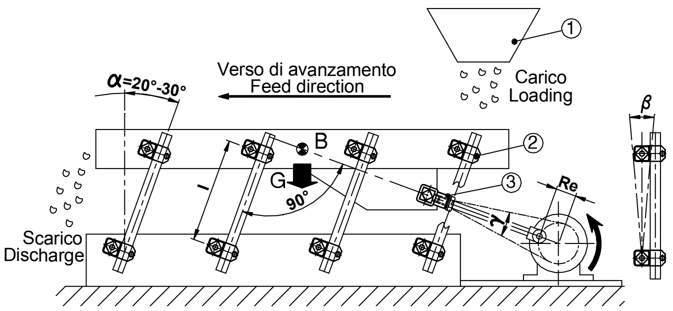
外部结构为烤炉涂漆，内部方管由 RAL 涂漆覆盖 / *The external body is oven-painted while the inner square is covered with a RAL varnish.*

使用 / DUTY

GF 振动组件主要应用于以连杆 / 曲柄制动的输送机和振动筛中具不可变轴距的弹性悬架。使用 GF 组件可为具单重块的系统或者具单重块和平衡重块的系统制造可变轴距的悬架。连接单位使用圆管，其费用由用户负担。

Oscillating components GF are generally used to realize rocker suspension in conveyor and screens actuated by a connecting rod/crank device. With GF components it is possible realize rocker suspension with adjustable axle base in one mass system or two mass system (with counter mass). The customer supplies the round connecting link that is realize with a round section tube.

应用 1 / *Application 1*:



图例说明:

1: 装料漏斗 / *Load hopper*

2: VIB GF 型弹性组件 / *GF Elastic component*

3: VIB TB 型弹性组件 / *TB Elastic Component*

B: 重心 / *Centre of gravity*

G: 重量 / *Weight* R_e: 曲柄半径 / *Crank radius*

α: 安装角度从 20°至 30° / *Rocker angle from 20° to 30°*

β: 工作角度 / *Working angle* γ: 曲柄振动角度 / *Oscillating crank angle*

l: 轴距 / *Distance between centers*

一个具单重块的振动组实例。

所用的计算步骤与在相关于 BT-F 段落中所描述的一致。

由两个 GF 弹性组件所组成的每个悬架的动力弹性 E_d 由以下关系式得到:

E_d: 动力弹性 = $\frac{M_d \cdot 360 \cdot 1000}{l^2 \cdot \pi}$ [N/mm]

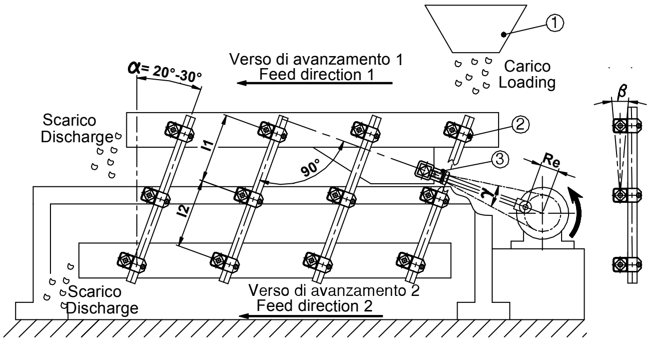
EXAMPLE OF A ONE-MASS VIBRATING UNIT.

The calculation diagram you should follow is as described in the BT-F paragraph.

Dynamic elasticity E_d for each suspension consisting of two elastic components GF is obtained from the relation:

E_d: *Dynamic elasticity* = $\frac{M_d \cdot 360 \cdot 1000}{l^2 \cdot \pi}$ *[N/mm]*

应用 2 / *Application 2*:



图例说明 / *Key*:

1: 装料漏斗 / *Load hopper*

2: VIB GF 型弹性组件 / *GF Elastic component*

3: VIB TB 型弹性组件 / *TB Elastic Component*

R_e: 曲柄半径 / *Crank radius*

α: 安装角度从 20°至 30° / *Rocker angle from 20° to 30°*

β: 工作角度 / *Working angle*

γ: 曲柄振动角度 / *Oscillating crank angle*

l_1: 上端渠轴距 / *Superior chute distance between centers*

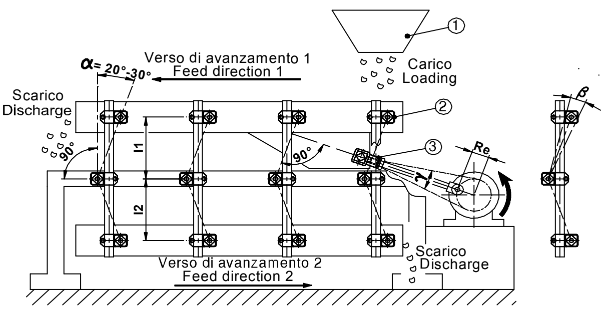
l_2: 下端渠轴距 / *Inferior chute distance between centers*

连接单位 (费用由用户负担)：推荐尺寸
CONNECTING LINK (to be supplied by the customer): RECOMMENDED DIMENSIONS

型号 *Type*	ØT	M_S	l_M	应用 *DUTY*
GF 40	30	3	160	仅适用于应用 1 - *Only application 1*
GF 40	30	4	220	应用 1 / 2 / 3 - *Application 1/2/3*
GF 40	30	3	300	应用 1 / 2 / 3 - *Application 1/2/3*
GF 50	40	3	200	仅适用于应用 1 - *Only application 1*
GF 50	40	4	250	应用 1 / 2 / 3 - *Application 1/2/3*
GF 50	40	5	300	应用 1 / 2 / 3 - *Application 1/2/3*

ØT: 连接管直径 / *Connecting tube diameter*
M_S: 管最低厚度 / *Minimum thickness*
l_M: 最大轴距 / *Maximum distance between centers*

应用 3 / *Application 3*:



图例说明 / *Key*:

1: 装料漏斗 / *Load hopper*

2: VIB GF 型弹性组件 / *GF Elastic component*

3: VIB TB 型弹性组件 / *TB Elastic Component*

R_e: Raggio della manovella / *Crank radius*

R_e: 曲柄半径 / *Crank radius*

α: 安装角度从 20°至 30° / *Rocker angle from 20° to 30°*

β: 最大工作角度 10° / *Working angle*

γ: 曲柄振动角度 / *Oscillating crank angle*

l_1: 上端渠轴距 / *Superior chute distance between centers*

l_2: 下端渠轴距 / *Inferior chute distance between centers*

具两份平衡重块的振动组实例（在渠槽上前进方向相同）。

所用的计算步骤与在相关于 TD-F 段落中所描述的一致。

由三个 GF 弹性组件所组成的每个悬架的动力弹性 E_d 由以下关系式得到:**E_d**: 动力弹性 = $\frac{270 \cdot M_d \cdot 1000}{\pi} \left(\frac{l_1^2 + l_2^2}{l_1^2 \cdot l_2^2}\right)$ [N/mm]

使用这项系统可能设计双重平衡振动槽。下端槽可用于将系统输送容量加倍，也可用于收集从上端槽落下的物料（筛子、校准器、除尘器等）。由上端渠槽输送的物料及由下端渠槽输送的物料的前进方向一致。

EXAMPLE OF A TWO-BALANCED-MASS VIBRATING UNIT (same feed directions on the channels).

The calculation diagram you should follow is as described in the TD-F paragrap.
Dynamic elasticity E_d for each suspension consisting of three elastic components GF is obtained from the relation:

***E_d**: Dynamic elasticity* = $\frac{270 \cdot M_d \cdot 1000}{\pi} \left(\frac{l_1^2 + l_2^2}{l_1^2 \cdot l_2^2}\right)$ *[N/mm]*

The above system can be used to make double balanced vibrating channels. The lower channel may be used to double the system conveyance capacity as well as to collect the material falling from the upper channel (sieves, calibrators, dusters, etc.). The feed direction of the material carried by the upper and lower channel is the same.

具两份平衡重块的振动组实例（在渠槽上前进方向相反）。

所用的计算步骤与在相关于 TD-F 段落中所描述的一致。

由三个 GF 弹性组件所组成的每个悬架的动力弹性 E_d 由以下关系式得到:**E_d**: 动力弹性 = $\frac{270 \cdot M_d \cdot 1000}{\pi} \left(\frac{l_1^2 + l_2^2}{l_1^2 \cdot l_2^2}\right)$ [N/mm]

使用这项系统可能设计双重平衡振动槽。上端槽和下端槽物料的前进方向相反。下端槽可用于将系统输送容量加倍，也可用于收集从上端槽落下的物料（筛子、校准器、除尘器等），并将其重新传送至装置起始端。为了获得在两个渠槽上相反的前进方向，悬架应与渠槽垂直安置，上端和下端 GF 弹性组件与在框架上固定的中央组件成 180°。

EXAMPLE OF A TWO-BALANCED-MASS VIBRATING UNIT (opposite feed directions on the channels).

The calculation diagram you should follow is as described in the TD-F paragraph.
Dynamic elasticity E_d for each suspension consisting of three elastic components GF is obtained from the relation:

***E_d**: Dynamic elasticity* = $\frac{270 \cdot M_d \cdot 1000}{\pi} \left(\frac{l_1^2 + l_2^2}{l_1^2 \cdot l_2^2}\right)$ *[N/mm]*

The above system can be used to make double balanced vibrating channels. The lower channel may be used to double the system conveyance capacity with opposite feed directions of the upper and lower channels as well as to collect the material falling from the upper channel (sieves, calibrators, dusters, etc.) in order to bring it to the starting point of the plant. The two channels opposite feed directions can be obtained by positioning suspensions perpendicular to the channels and by rotating of 180° the upper and lower GF elastic components with respect to the central component which is fixed to the structure.